

Cooling plate for metallurgical furnaces

The invention relates to a cooling plate, consisting of copper or low-alloy copper alloy, for metallurgical furnaces provided with an outer furnace casing plate, having at least one, preferably at least two, coolant passages which run inside the cooling plate, coolant pipe sections for coolant to flow in and out being led to the outside through the furnace casing plate.

Cooling plates of this type are arranged between the shell and the lining and are connected to a cooling system. On the side facing the interior of the furnace, the cooling plates are in part provided with refractory material.

DE 39 25 280 has disclosed a cooling plate in which the cooling passages are formed by pipes cast into cast iron. The bearing lug is also connected to the cooling system. The dissipation of heat from these plates is low on account of the low thermal conductivity of the cast iron and on account of the resistance between the cooling pipes and the plate body caused by a layer of oxide or an air gap.

In the event of a loss of the blast furnace refractory lining after a certain operating time, the inner surface of the cooling plates is directly exposed to the temperature of the furnace. Since the furnace temperature is well above the melting point of cast iron and the inner heat transfer resistances of the cooling plates lead to unsatisfactory cooling of the hot side of the plates, accelerated wear of the cast iron plates is inevitable and the service life is correspondingly limited.

DE 199 43 287 A1 has disclosed a copper cooling plate which, in the vicinity of the upper coolant pipe sections, is fixedly connected to the furnace casing by

means of a fixed-point securing element. In addition, the upper coolant pipe sections are likewise fixedly connected to the furnace casing. Further securing elements are designed as movable-point securing
5 elements which allow mobility in both the horizontal direction (x) and in the vertical direction (y). The lower coolant pipe sections are connected in a gastight manner to the furnace casing only by means of standard compensators. Therefore, in this region the cooling
10 plate is not fixed in any of the three spatial directions.

On account of the fact that the side of the cooling plate which faces the interior of the furnace reaches
15 temperatures of more than 300°C and the side which faces the furnace casing remains at coolant temperature, i.e. approximately ambient temperature, the cooling plate is exposed to very high thermally induced stress forces. In the cooling plate described
20 in DE 199 43 287 A1 this means that the plate, which is of course completely fixed at a number of locations, is plastically deformed under the influence of heat and during cooling bulges into the interior of the furnace in the shape of a dish. This leads to cracks in the
25 coolant pipe sections and to coolant leaks.

In view of the prior art, the invention is based on the object of providing a cooling plate in which dish-shaped deformation of this type is no longer possible
30 and in which, therefore, stress-induced cracks no longer occur in the coolant pipe sections.

According to the invention, this object is achieved through the fact that the cooling plate is provided
35 with holding pipes which are led to the outside through the furnace casing plate and which, after they have been passed through the furnace casing plate, are provided with securing elements, in particular holding plates or holding disks, and the holding pipes and the

securing elements being made from a material which has an increased strength compared to copper or low-alloy copper alloy.

5 The cooling plate according to the invention is movable in both the vertical direction (y) and the horizontal direction (x) at the positions of the holding pipes, whereas a movement in the z direction, i.e. "dishing" in the direction of the interior of the furnace, is
10 prevented by the securing elements which are arranged at the holding pipes and are supported against the furnace casing outside the latter.

Unlike in the prior art, any stresses which
15 nevertheless occur in the z direction, are not borne by the copper coolant pipe sections, which are not suitable for this purpose, but rather by the holding pipes and securing elements, which are made from a much more suitable material.

20 The material which is specifically preferred for the holding pipes and securing elements is steel, possibly even special steel. However, in principle any material is suitable, provided that it satisfies the requirement
25 of having a significantly higher strength than copper or low-alloy copper alloy, as is the case with steel. A material which can be welded at least to itself and preferably also to copper or low-alloy copper alloy, as is likewise the case with the particularly preferred
30 material steel, is also preferred.

According to an advantageous embodiment, the cooling plate according to the invention is connected to the furnace casing plate in a central region by means of a
35 fixed-point securing element.

A fixed-point securing element of this type may, for example, be a securing bolt and at this location fixes the plate in each of the three spatial directions.

In this way, the cooling plate according to the invention is fixed in position and cannot be moved out of this position in any way by thermally induced stress forces. On the other hand, thermal expansion of the plate remains unrestricted from this central fixed point.

According to a further advantageous embodiment, the cooling plate - in particular with a cooling plate height/width ratio of ≥ 3 - is provided with at least one movable-point securing element which is arranged above and/or below the fixed-point securing element and allows mobility only in the vertical direction.

Alternatively, the cooling plate - in particular with a cooling plate height/width ratio of < 3 , preferably < 2 - is provided with at least one moveable-point securing element which is arranged to the left and/or to the right of the fixed-point securing element and allows mobility only in the horizontal direction.

A movable-point securing element of this type may, for example, be a securing bolt with disk, the disk remaining unwelded to the furnace casing plate and being able to slide in a guide in one direction, and the movable-point securing element, depending on its orientation, fixes the plate either in the x direction or in the y direction and at any rate in the z direction.

In addition to the fixing achieved by the central fixed point, both the variants described ensure that the cooling plate is not simply completely fixed in position, but rather that the cooling plate is also prevented from twisting as a result of thermally induced stress forces. Furthermore, thermal expansion of the plate in the directions allowed by the movable-point securing elements is freely possible.

A particularly preferred embodiment of the cooling plate according to the invention consists in the cooling plate having tongues and grooves on the side
5 which faces the interior of the furnace, the tongues being segmented in their longitudinal direction.

The tongues are the part of the cooling plate which is least exposed to the cooling action of the coolant.
10 Consequently, the tongues are the part of the cooling plate as a whole which reaches the highest temperature (the temperature of approximately 300°C mentioned in the introduction). Dividing the tongues into individual sections reduces the stress forces caused by the
15 thermal expansion of the tongues to the minimum possible. Subdividing the tongues alone would in cooling plates known from the prior art be a suitable way of reducing the "dishing" of the cooling plates and increasing the service life of the cooling plates, in
20 particular the coolant pipe sections.

In order not to adversely affect the mechanical strength of the cooling plate as a whole, it is in this context preferable for the segmenting of the tongues to
25 be implemented as incisions arranged substantially at right angles to the tongues.

Furthermore, these incisions are preferably implemented in such a manner that they are not arranged above the
30 coolant passages, but rather between them. In the event of any cracks being formed, the risk of further cracking into the coolant passages can be reduced in this way.

35 In order not to adversely affect the load-bearing capacity of the tongues - which usually run horizontally - it is preferable for the segmenting of the tongues to be implemented in such a way that the

individual segments are horizontally offset from one another.

5 Various further embodiments relate to the detailed design of the holding pipe, securing element and coolant pipe section. These particular embodiments form the subject matter of claims 6 to 12.

10 In principle, it is preferably for a holding pipe to in each case surround a coolant pipe section, i.e. for a holding pipe to be led to the outside through the furnace casing, and for a coolant pipe section in each case to be lead to the outside through the furnace casing inside a holding pipe.

15 The nature of the connection between holding pipe and cooling plate or between coolant pipe section and cooling plate or even between holding pipe and coolant pipe section may be of varying design.

20 According to a first and preferred variant, a disk-like connecting piece is welded to the holding pipe which surrounds the coolant pipe section, and this connecting piece is screwed to the cooling plate. The coolant pipe section is welded to the cooling plate.

30 According to a second variant, the holding pipe which surrounds the coolant pipe section is directly welded to the cooling plate, and the coolant pipe section is also welded to the cooling plate.

35 According to a further variant, a connecting piece in the form of a disk or ring is introduced between the coolant pipe section and the holding pipe. The coolant pipe section and the holding pipe are seated on this connecting piece. The connection between the connecting piece and the cooling plate, on the one hand, and between the connecting piece and the coolant pipe

section or the holding pipe, on the other hand, is preferably effected by welding.

According to a further variant, the coolant pipe
5 section is designed as a single part and is provided with a flange, which flange is secured to the cooling plate. In this case, the holding pipe surrounds the coolant pipe section, is seated on this flange and is secured to it by welding.

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According to a further variant, a holding pipe is also designed as a coolant pipe section, in which case both functionalities, i.e. the supply and removal of coolant and the holding function, are performed by this one
15 pipe section.

With the exception of the last variant described, in all embodiments the coolant pipe sections may be made either from the same base material as the cooling plate
20 or from a different material, preferably the material of the holding pipe.

For the last variant described, this range of choices does not exist, since in this case the coolant pipe is
25 simultaneously also the holding pipe and therefore must in any case be made from the material of the holding pipe.

The invention is explained in more detail below in the
30 drawings presented in Fig. 1 to Fig. 9.

Fig. 1 shows a two-passage cooling plate

Fig. 2 shows a four-passage cooling plate

Fig. 3 shows an arrangement of a plurality of cooling
35 plates

Fig. 4 shows the segmenting of a four-passage cooling plate

Fig. 5-9 show various designs of holding pipe

Fig. 1 shows a two-passage cooling plate 1 which is secured to a furnace casing plate 2. The cooling plate consists of copper and has tongues 3 on the side facing the interior of the furnace. The space between cooling plate 1 and furnace casing plate 2 is backfilled with refractory material 4. Further cooling plates 1' are arranged above and below and - not shown - to the sides of the cooling plate 1. The cooling plate 1 is provided with vertically running cooling passages 5, which are designed as blind bores in the cast or rolled plate body. Coolant pipe sections 6 for supplying and removing coolant (usually water) are led through the furnace casing 2 at the upper and lower ends of each cooling passage 5. At each coolant pipe section 6, a holding pipe 7 - surrounding the coolant pipe section 6 - is likewise led to the outside through the furnace casing. The holding pipe 7 is screwed to a disk-like connecting piece 8, which for its part is secured to the cooling plate 1 by screw connection 9. Outside the furnace casing 2, the holding pipe 7 is provided with a welded-on holding disk 10 which limits the mobility of the cooling plate 1 in the direction of the interior of the furnace. Holding pipe 7 and coolant pipe section 6 are connected in a gastight manner to the furnace casing plate 2 by means of a standard compensator 11. In the center of the cooling plate 1, the latter is fixedly connected to the furnace casing plate 2 by means of a fixed-point securing elements 12 designed as a securing bolt. The fixed-point securing elements 12 is welded in a gastight manner to the furnace casing plate 2. Movable-point securing elements 13 are arranged above and below the fixed-point securing elements 12. The movable-point securing elements 13 are likewise designed as securing bolts, but are not welded in a gastight manner to the furnace casing plate 2, but rather can slide up and down in a guide 14. To provide a seal with respect to the interior of the furnace, sealing hoods 15 are arranged over the movable-point securing elements 13.

Fig. 2 shows a four-passage cooling plate 16 which, apart from having twice the number of cooling passages 5, is substantially identical to the cooling plate 1 illustrated in Fig. 1. On account of the different height/width ratio, however, the movable-point securing elements 13 are not arranged above and below the fixed-point securing elements 12, but rather are in each case arranged laterally with respect to the latter. The guides 14 for the movable-point securing elements 13 are arranged in such a way that they can slide in the horizontal direction.

Fig. 3 diagrammatically depicts the arrangement of two-passage cooling plates 1 and four-passage cooling plates 16 in a furnace. The figure also illustrates the system of coordinates which indicates the x, y and z directions to which the text has repeatedly referred.

Fig. 4 shows the tongues 3 of the cooling plate 16 which are segmented in the horizontal direction. The individual segments are in each case of approximately identical length and are horizontally offset by approximately half their length.

Fig. 5 shows an enlarged illustration of the preferred variant of the design of holding pipe 7, coolant pipe section 6 in accordance with the invention and also the way in which the connecting piece 8 is secured to the cooling plate 1 by means of screw connection 9.

The design shown in Fig. 6 differs from that shown in Fig. 5 in that the connection between holding pipe 7 and cooling plate 1 is produced by welding.

Fig. 7 shows an embodiment in which both the holding pipe 7 and the cooling pipe section 6 are secured to the connecting piece 8.

Fig. 8 illustrates a coolant pipe section 6 which is designed as a single part and is provided with a flange, the holding pipe 6 also being secured to this flange.

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Fig. 9 shows a special embodiment. On an annular connecting piece 8 which is welded to the cooling plate 1 there is fitted a pipe section 17 which is welded to the connecting piece 8, the pipe section 17 being made from the same material as the holding pipes, i.e. for example steel, and, on account of the higher strength compared to copper, simultaneously serving as both the coolant pipe section and the holding pipe.

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15 In all the drawings represented in Fig. 5 to 9, holding disks 10, which are arranged directly outside the furnace casing 2 and are welded to the holding pipes 7, 17, fix the corresponding cooling plate 1 in the z direction with respect to the furnace interior.